Numerical analysis of the stress and displacement level caused by axial load P=15000 N in a car scissor jack BD-02B2

1. INTRODUCTION

The scissor jacks are simple and basic mechanisms to operating at the car suspension. Thanks to them, it is possible to both independent replacement of worn suspension components, and repair of faults occurring in the suspension.

Cutting-edge research techniques strength of materials, provide analysis of mechanisms already working on the way their design. Finite element method allows to conduct numerical analysis designed components/mechanisms, already prior to them production. This method is beneficial to improve the properties of construction elements and also to reduce the applied load, which adversely affect long-term and correct operation of the system.

The paper analyzes the numerical level of displacements and stresses that occur in critical places the scissor jack.
The aim of the numerical calculations was to determine the stress distribution in the model, for the vertical force $P = 15000$ N, which is the maximum load that can move the scissor jack. The resulting reduced stress distributions, consistent with the endurance hypothesis HMH (Huber-Mises-Hencky), were compared with the material properties of the jack. Numerical calculations were carried out to determine the strength of the mechanism.

In the literature reference [9, 10] the author deals with an introduction to the interface and a description of the program Abaqus and basic approach to design parts. The author also takes into account the approach to linear statics and to generate an appropriate mesh type to designed objects.

In the papers [4, 5] the authors deal with the presentation of a general approach to the problem of testing the strength of materials, through numerical modeling in Abaqus environment. The authors explain the idea of the work structure design based on the study of stability.

Through publications [11, 12, 13] it is possible to understand the Catia environment and the practical use of knowledge on how to design and generate simulation kinematic structure.

In references [2, 7, 8] the authors dealt with issues related to research vehicle lifts, which is the necessary basis for the development of the issues described in the present article. The publication [3] the author presents the problem of compression of thin-walled profiles are susceptible to loss of stability, which is close to a submitted article.

In [6] the authors take up the subject to generate 3D objects from 2D drawings prepared in order to increase the potential of construction work and to improve the design of machinery.

2. MATERIALS AND METHODS

The subject of research was scissor jack BD-02B2. The mechanism is dedicated to passenger vehicles with an unladen weight not exceeding 1500 kg.

In fact the lifting at the same time is one of the sides of car, so load is approximately only half the weight of the vehicle. Initial figure of model was designed in Catia V5R20 environment in which the simulation was also conducted kinematic, showing the correct operation of the jack and range of mobility. Numerical analysis was made using Abaqus 6.10. The mechanism was analyzed numerically, at maximum lift height of 345 mm.

The jack is designed on the basis of measurements taken on a real object. The spatial model designed in CATIA, and the actual form of the scissor jack, shown in Figure 1.
The object of research was characterized by a uniform material properties. The mechanism is designed in accordance with the characteristics of the structural steel C45.

The material of which made the scissor jack with its basic features of the material are shown in Table 1.
Tab. 1. Characteristics of the steel [1]

<table>
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<th>Material: Steel C45</th>
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<tr>
<td>Young’s Modulus [Mpa]</td>
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<td>Poisson’s Ratio</td>
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<tr>
<td>Yield Strength [Mpa]</td>
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<td>Tensile Strength [Mpa]</td>
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Numerical calculations were based solely on static calculations, so the mass of the mechanism were not taken into account. Any interactions prevailing in the system, were related only with contact on the tangential and normal direction with taking into account the coefficient of friction of 0.15.

The work of mechanism is performed by applying a vertical load to the crown, which acting in a downward direction in accordance with the X axis, and the mechanism undergoes axial compression.

Boundary conditions were associated with total restraint of base of the scissor jack, as well as the application of force concentrated at the reference point, which is coupled with the upper planes of the crown of scissor jack, through use coupling constraint in the interaction module, as shown in Figure 2.

![Fig. 2. Boundary conditions [source: own research]](image-url)
Proper conduct numerical simulation was made possible by assigning appropriate FEM mesh model. For this aim was used a tetrahedral elements called C3D4 and eight-node elements C3D8R with a reduced number of points of integration. Reduced integration technique removes the problem of irregular form of the deformation of objects. Eliminated are the components of higher order polynomials, which is beneficial to improve the results of numerical analysis performed [14]. Finite element mesh was composed of more than 76000 items.

The bolt, nut and screw guide element, are part which have a tetragonal structure of mesh with a greater density, while all other components are assigned only type hexagonal type of mesh by using the appropriate partitioning methods. Visualization of the generated mesh is shown in Figure 3.

3. RESULTS

FEM analysis made it possible to obtain a reduced stress distribution H-M-H in the investigated mechanism. The most loaded part of the mechanism is screw guide element, in which the stress level was about 504 MPa. The remainder of the mechanism to give the symmetric nature of stress distribution.

Analysis showed a lack of exceed of strength border components, so the system will not be exposed to permanent damage. The level of stresses and displacements shown in Figure 4.
Fig. 4. The results of numerical analysis: a) stress distribution in scissor jack, b) displacements occurring levels, c) maximum stress level [source: own research]
Only the screw guide element and nut have a stress exceeding the yield strength. In these parts there was a maximum of stress concentration occurred at the connection of these elements with arms of the scissor jack as shown in Figure 4c.

The other components are not exposed to any loss of ductility. In fact, the emphasis generated by the weight of the vehicle, would be much lower, because is raised only a part of the vehicle, so the level of stress would have much less.

The paper presents the results of the analysis only with the maximum allowable load equal 15000 N and at maximum lift height equal to 345 mm. Any numerical calculation was carried out taking into account the issues of non-linear geometrically using the Newton-Raphson method.

4. CONCLUSIONS

On the basis of the numerical analysis can draw the following conclusions:
– analysis of the stress distribution allows to determine the sensitive zones of mechanisms, which have the greatest concentration of stress in the individual components and structural elements;
– FEM analysis results showed no cross the border strength of the material, confirming the correctness of the designed system;
– numerical research methods make it possible to further optimize the components in terms of reducing the concentration of stresses and strains.

In this study, showed no exceeding the maximum permissible strength of the material from which made the scissor jack. Only elements connecting the arms of scissor jack showed stress exceeding the yield strength at a given maximum emphasis.

Application of FEM to analyze the stress distribution in the mechanisms subjected to loads, allows to take stress and strain state of this type of construction, which nowadays is indispensable in engineering applications. Numerical analyzes are now an essential tool in the market when tested the strength of materials, so it is possible to prevent an unnecessary manufacturing errors.

REFERENCES